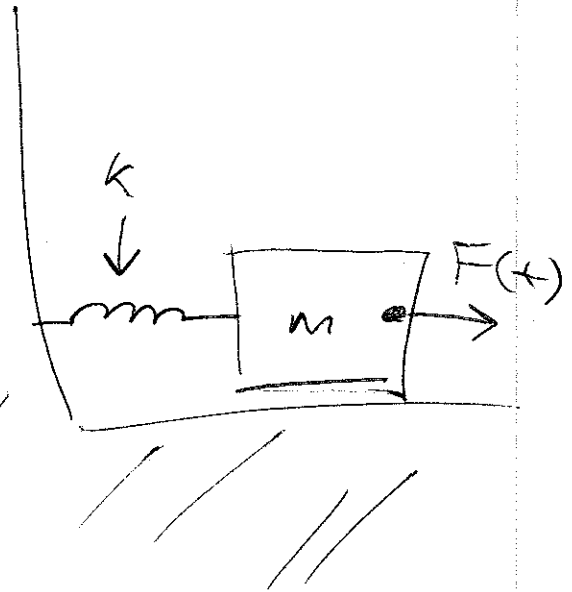


Recap of Forced Oscillator

Let $F(t) = F_0 e^{i\omega_F t}$

$\omega_F =$ forcing frequency, a parameter we can choose.



Eg. of Motion:

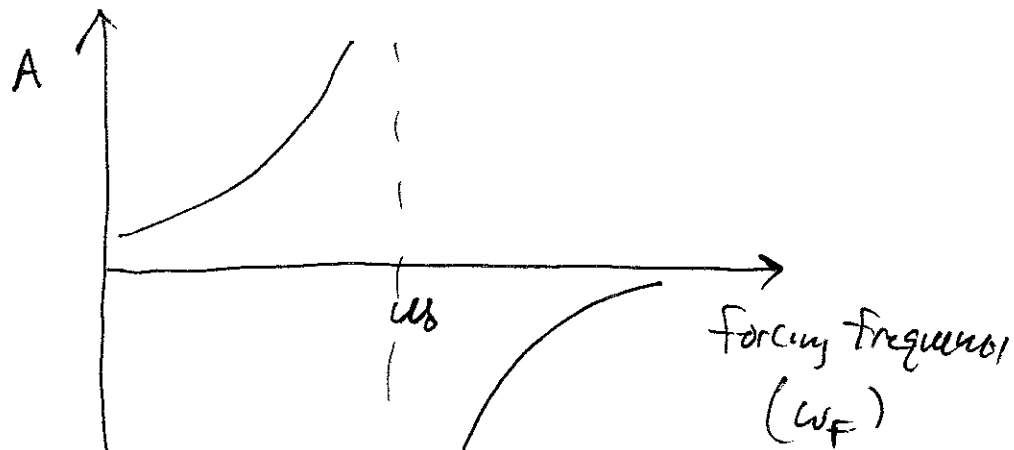
$\ddot{x} + \omega_0^2 x = \frac{F_0}{m} e^{i\omega_F t}$, where $\omega_0 = \sqrt{\frac{k}{m}} =$ "natural" frequency

↑ actual frequency ↑ forcing frequency.

Solution:

$x(t) = A e^{i(\omega_F t + \delta)}$
 where $A = \frac{F_0}{m(\omega_0^2 - \omega_F^2)}$
 and $\delta = \phi$

A & δ are not free parameters, they are fixed. In particular, A depends on the forcing frequency:



negative A means
a phase shift of 180° .
($-1 = e^{i\pi} = e^{i(180^\circ)}$)

Consequences:

- ① If $\omega_f \ll \omega_0$, then A is small and positive \Rightarrow oscillator moves 100% in phase with the external force.
- ② If $\omega_f \gg 2\omega_0$, then A is small and negative \Rightarrow oscillator moves 180° out-of-phase with the external force.
- ③ If $\omega_f \approx \omega_0$, then A becomes very large \Rightarrow Resonance.

If you force an oscillator to go at its natural frequency, the oscillator response will be large, which is a resonant phenomena.

Compare to the Simple Harmonic Oscillator (No Forcing)

- Always goes at ω_0 .
- A & δ are ~~free~~ free parameters which we can choose in order to satisfy the initial conditions.

For the Forced Oscillator, the initial conditions affect the transient (short-term) behavior. We will study this later. The solution we are studying now is the long-term behavior.

Simple Harmonic Oscillator with Damping.

Real mechanical oscillators always have some resistive force which is non-conservative.

Resistive forces convert mechanical energy to heat. They must be modeled empirically.

~~We~~ A very simple model is:

$$F_{\text{resistive}} = -b v = -b \dot{x}$$

minus sign means the force acts opposite the velocity.
 Constant b the oscillator's velocity

This model works reasonably well when v is not too ~~for~~ large

Eq. of Motion:

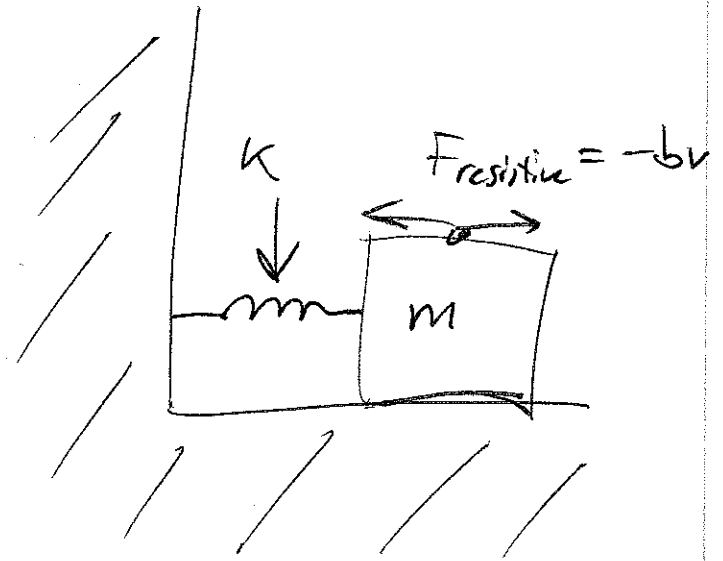
$$-kx - bv = m\ddot{x}$$

$$\uparrow$$

$$v = \dot{x}$$

$$\ddot{x} + \frac{b}{m}\dot{x} + \frac{k}{m}x = 0$$

\uparrow
no driving
force



Define: $\gamma \equiv \frac{b}{m}$. Also, $\omega_0^2 \equiv \frac{k}{m}$. Then

$$\boxed{\ddot{x} + \gamma\dot{x} + \omega_0^2 x = 0} \quad \text{Eq. of Motion.}$$

\Rightarrow small γ means very little drag.

\Rightarrow large γ means large drag.

Guessed Solution:

$$x(t) = A e^{i(\omega_d t + \delta)}$$

ω_d = "frequency at which the damped oscillator goes"
(not yet known!).

Also, A & δ are not known either.

Substitute the guess:

$$\ddot{x} = -\omega_d^2 x, \quad \dot{x} = i\omega_d x$$

Result:
$$-\omega_d^2 x + i\omega_d \dot{x} + \omega_0^2 x = 0$$

$x(t)$ divides out:

$$-\omega_d^2 + i\omega_d \gamma + \omega_0^2 = 0$$

A purely algebraic Equation for ω_d in terms of γ and ω_0 .

Notice That

- ① $\omega_d =$ purely real cannot satisfy this equation.
- ② $\omega_d =$ purely imaginary also cannot satisfy this equation.

We must allow ω_d to have both real & imaginary parts:

$$\omega_d = \omega_r + i\omega_i$$

\uparrow real part \uparrow imaginary part.

So our equation for ω_d says:

~~$$-(\omega_r + i\omega_i)^2 + i(\omega_r + i\omega_i)\gamma + \omega_0^2 = 0$$~~

Gathering real & imaginary terms we have

① $-\omega_r^2 + \omega_i^2 - \omega_i \gamma + \omega_0^2 = 0$ ← real terms

② $i(-2\omega_r \omega_i + \omega_r \gamma) = 0$ ← imaginary terms must add to zero.

must add to zero

From (2) we must have:

$$\boxed{\omega_i = \frac{\gamma}{2}}$$

Then substitute into (1):

$$\boxed{\omega_r^2 = \omega_0^2 - \frac{\gamma^2}{4}}$$

Therefore: $\omega_d = \left(\omega_0^2 - \frac{\gamma^2}{4}\right)^{1/2} + i\left(\frac{\gamma}{2}\right)$

So our guessed solution becomes:

$$x(t) = \cancel{Ae^{i(\omega_d t + \delta)}} Ae^{i(\omega_d t + \delta)}$$

$$= Ae^{i(\omega_r + i\omega_i)t + \delta}$$

$$= Ae^{-\omega_i t} e^{i(\omega_r t + \delta)}$$

$$\boxed{x(t) = Ae^{-\gamma t/2} e^{i(\omega_r t + \delta)}, \text{ where } \omega_r = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$

We still have 2 unknown parameters: A & δ .

These will be determined by initial conditions.

Comment

① If we set $\gamma = 0$ (no damping), then we recover the SHO solution: $x(t) = Ae^{i(\omega_0 t + \delta)}$

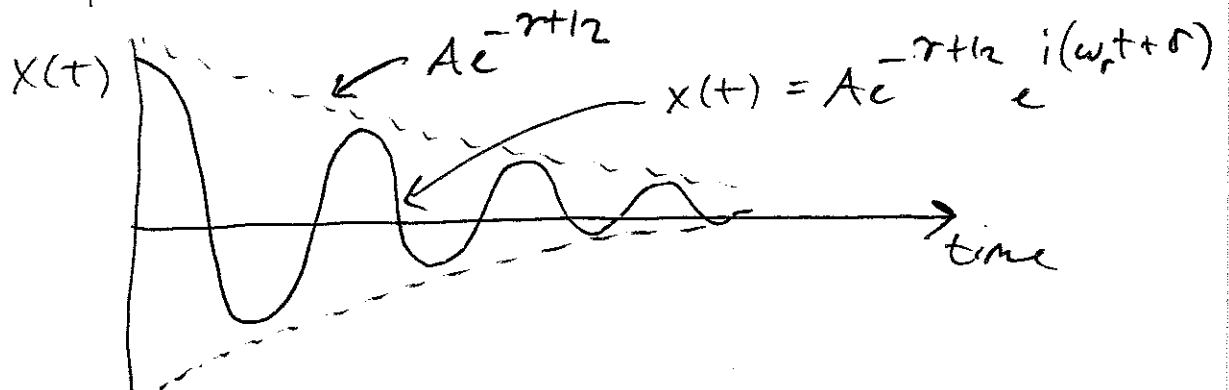
- (2) For non-zero γ , we have an oscillation at a frequency of

$$\omega_r = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$

which is not the natural frequency due to the $\frac{\gamma^2}{4}$ term.

\Rightarrow The damped oscillator goes at a frequency which is ~~slightly~~ smaller than the natural frequency. If the damping coefficient is small, then the damped oscillator frequency will only be very slightly less than ω_0 .

- (3) The amplitude of the oscillation decays exponentially in time:



The mechanical energy of the oscillator is being converted into heat, and the amplitude is decreasing.

Energy Considerations

Mechanical energy = $E \equiv$ Kinetic Energy + Potential Energy

$$E = KE + U$$

For a mass on a spring, $U = \frac{1}{2}kx^2$

$$KE = \frac{1}{2}m\dot{x}^2$$

$$\therefore E = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}kx^2$$

As a function of time,

$$x(t) = A \cos(\omega t + \delta), \text{ and } \dot{x}(t) = -A\omega \sin(\omega t + \delta)$$

$$\text{Also, } \omega_0^2 = k/m \Rightarrow k = m\omega_0^2.$$

And in the case of no damping, $\omega = \omega_0$.

$$\begin{aligned} \therefore E(t) &= \frac{1}{2}mA^2\omega_0^2 \sin^2(\omega_0 t + \delta) + \frac{1}{2}m\omega_0^2 A^2 \cos^2(\omega_0 t + \delta) \\ &= \frac{1}{2}m\omega_0^2 A^2 \underbrace{(\sin^2(\omega_0 t + \delta) + \cos^2(\omega_0 t + \delta))}_1 \end{aligned}$$

$$E(t) = \frac{1}{2}m\omega_0^2 A^2 = \text{constant (no damping)}$$

Energy is conserved.

But suppose the oscillator is "lightly damped":

$$\text{Then } F_{\text{resistive}} = -\underset{\substack{\uparrow \\ \text{small}}}{b}\dot{x} \text{ and } \gamma \equiv \frac{b}{m}$$

$$\begin{aligned} \text{Then } x(t) &= \text{Re} \left[A e^{-\gamma t/2} e^{i(\omega_r t + \delta)} \right] \\ &= A e^{-\gamma t/2} \cos(\omega_r t + \delta) \end{aligned}$$

$$\text{where } \omega_r = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}}$$

We can write

$$x(t) = \underbrace{Ae^{-\gamma t/2}}_{A(t)} \cos(\omega t + \delta)$$

$$x(t) = A(t) \cos(\omega t + \delta), \text{ and } \omega_d \approx \omega_0 \text{ in the case of light damping.}$$

Then the total mechanical energy is

$$E(t) = \frac{1}{2} m \omega_0^2 A^2 = \frac{1}{2} m \omega_0^2 (A(t))^2 \\ = \frac{1}{2} m \omega_0^2 (Ae^{-\gamma t/2})^2$$

$$E(t) = \underbrace{\frac{1}{2} m \omega_0^2 A^2}_{E_0} e^{-\gamma t}$$

Call this E_0

For a lightly damped oscillator,

$$\rightarrow \boxed{E(t) = E_0 e^{-\gamma t}}$$

E decays away

exponentially with

time constant $\left(\frac{1}{\gamma}\right)$. (γ has units of $\frac{1}{\text{seconds}}$, so $\frac{1}{\gamma}$ has units of time)

"Q Factor" or "Quality Factor"

We want to define a quantity which tells us whether the oscillator loses energy quickly or slowly.

- "high Q" = "high quality" = low energy loss rate
- "low Q" = "low quality" = low energy loss rate

We define it this way:

Question: What fraction of the energy is lost in the time it takes for the oscillator's phase to advance by 1 radian?

Call this amount of time t_1 . By definition

$$\underbrace{\omega_0 t_1}_{\substack{\text{phase} \\ \text{change} \\ \text{in time } t_1}} = 1 \quad \Rightarrow \quad \boxed{t_1 = \frac{1}{\omega_0}} \quad \begin{array}{l} \text{Time for} \\ \text{phase to} \\ \text{change by} \\ 1 \text{ radian} \end{array}$$

(Recall that $\cos(\underbrace{\omega_0 t + \phi}_\text{phase change})$)

Answer: The fraction of energy which remains after time t is

$$\frac{E(t)}{E_0} = e^{-\gamma t} \approx 1 - \gamma t \quad \begin{array}{l} \text{for small } \gamma \text{ and} \\ \text{small } t. \end{array}$$

∴ Fraction of energy lost is $\boxed{\gamma t}$.

Therefore in time $t_1 = \frac{1}{\omega_0}$, the fraction lost is

Fraction lost in time t_1 . $\rightarrow \boxed{\gamma \left(\frac{1}{\omega_0}\right)}$

We define the "Q factor" to be the inverse of this fraction:

$$\frac{\gamma}{\omega_0} \equiv \frac{1}{Q}$$

↑ "Quality factor"
or "Q factor"

or $\boxed{Q \equiv \frac{\omega_0}{\gamma}}$ For a lightly damped oscillator.

Ex: IF $Q = 100$, then the damped oscillator loses $\frac{1}{Q} = 1\%$ of its energy in time $t_1 = \frac{1}{\omega_0}$.

EX: IF $Q = 1000$, then the damped oscillator loses $\frac{1}{Q} = 0.1\%$ of its energy in time $t_1 = \frac{1}{\omega_0}$.

Other ways to think about Q:

$$Q \equiv \frac{1}{\text{Fraction of energy lost in time } t = \frac{1}{\omega_0}}$$

We can re-write it as

$T = \text{period} = \text{time for one complete cycle}$

$$T = \frac{1}{f} = \frac{2\pi}{\omega_0}$$

∴ Fraction of energy lost in one period = γT
 $= \gamma \left(\frac{2\pi}{\omega_0} \right)$

$$= \frac{2\pi}{(\omega_0/\gamma)}$$

Fraction of energy lost in one period = $\frac{2\pi}{Q}$

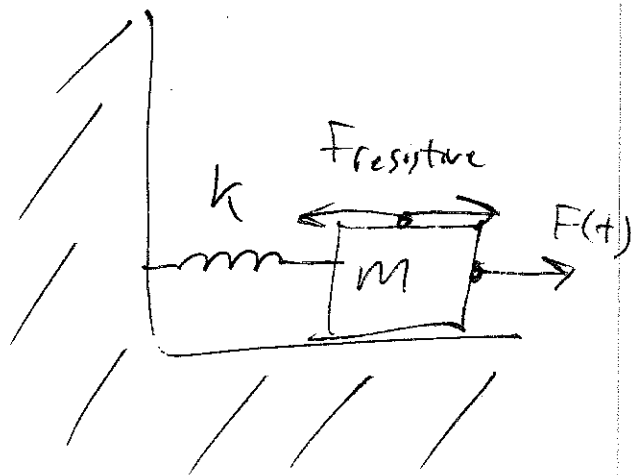
Forced Oscillator with damping

A harmonic oscillator with both damping and a driving force.

Eg. of Motion:

$$\ddot{X} + \gamma \dot{X} + \omega_0^2 X = \frac{F_0}{m} e^{i\omega_f t}$$

↑ damping
↑ natural frequency
↑ forcing frequency



Guessed Solution: The forced oscillator with no damping oscillates at the forcing frequency.

Let's try the same solution for the damped case:

$$X(t) = A e^{i(\omega_f t + \delta)}$$

↑ forcing frequency
← Guessed Solution

The usual question is: what are A and δ ?

Substitute: $\ddot{x} = -\omega_F^2 x$, $\dot{x} = i\omega_F x$

$$\Rightarrow -\omega_F^2 x + i\omega_F \tau x + \omega_0^2 x = \frac{F_0}{m} e^{i\omega_F t}$$

$\uparrow \qquad \qquad \qquad \uparrow \qquad \qquad \qquad \uparrow$
 $\left[A e^{i(\omega_F t + \delta)} \right]$

So $e^{i\omega_F t}$ cancels everywhere:

$$\left[-\omega_F^2 + i\omega_F \tau + \omega_0^2 \right] A e^{i\delta} = \frac{F_0}{m}$$

$$A (\omega_0^2 - \omega_F^2) + i(\omega_F \tau) A = \frac{F_0}{m} e^{-i\delta}$$

A complex number
in cartesian form

A complex number
in polar form.

We have a Real eq. and an imaginary eq.:

$$\boxed{A(\omega_0^2 - \omega_F^2) = \frac{F_0}{m} \cos(-\delta) = \frac{F_0}{m} \cos \delta} \quad \textcircled{1} \text{ Real Eq.}$$

and $\boxed{A\omega_F \tau = \frac{F_0}{m} \sin(-\delta) = -\frac{F_0}{m} \sin \delta} \quad \textcircled{2} \text{ Imag. Eq.}$

Take the ratio of the two equations to eliminate A :

$$\frac{\textcircled{2}}{\textcircled{1}} : \frac{\omega_F \tau}{(\omega_0^2 - \omega_F^2)} = \frac{-\sin \delta}{\cos \delta} = -\tan \delta$$

$$\delta = \text{phase shift of } x(t) \text{ relative to the forcing function} = \tan^{-1} \left[\frac{-\omega_F \tau}{\omega_0^2 - \omega_F^2} \right]$$

Phase Shift of the forced oscillator with damping

$$\delta(\omega_F) = -\tan^{-1} \left[\frac{\omega_F \tau}{\omega_0^2 - \omega_F^2} \right]$$

phase shift depends on the forcing frequency.

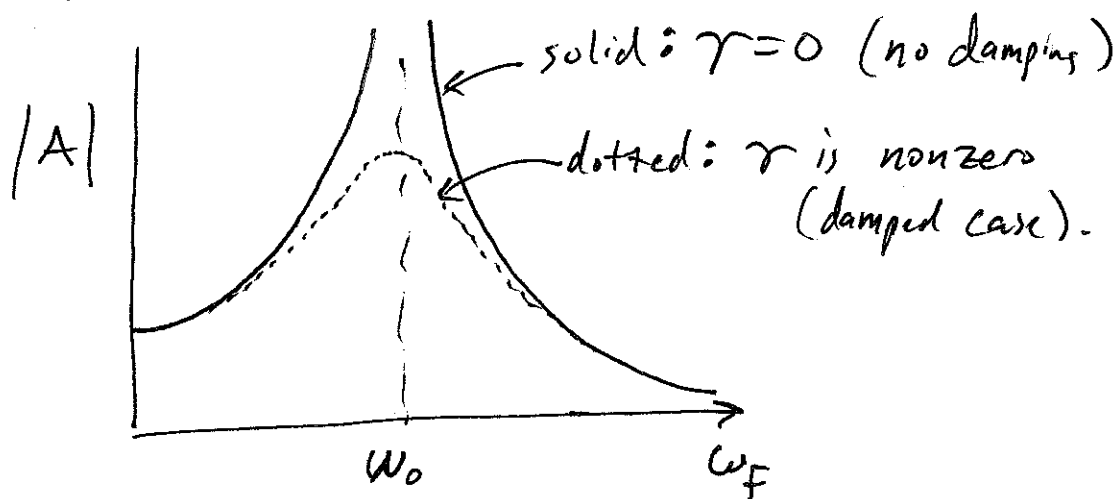
Also, we can solve for A by squaring (1) & (2) and adding:

$$\textcircled{1}^2 + \textcircled{2}^2 \Rightarrow A^2 \left[(\omega_0^2 - \omega_F^2)^2 + (\omega_F \tau)^2 \right] = \left(\frac{F_0}{m} \right)^2 \underbrace{[\cos^2 \delta + \sin^2 \delta]}_1$$

Amplitude of the forced oscillator with damping, as a function of the forcing frequency.

$$A(\omega_F) = \frac{F_0/m}{\sqrt{(\omega_0^2 - \omega_F^2)^2 + (\omega_F \tau)^2}}$$

Just like the forced oscillator with no damping, we have a ~~is~~ resonance where A becomes large when ω_f is chosen to be near ω_0 :

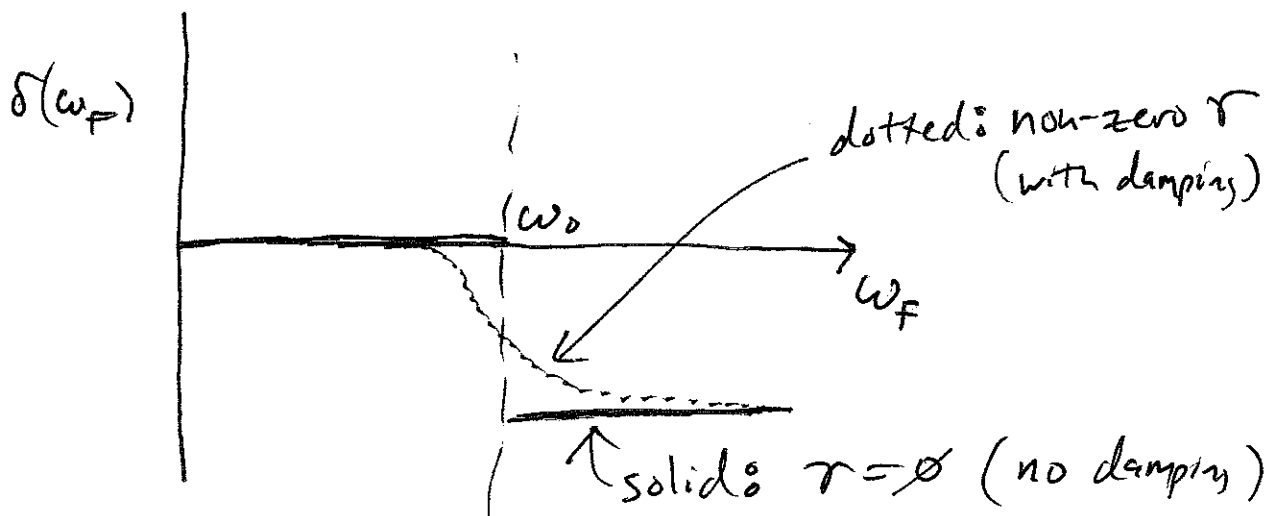


The damping coefficient γ removes the infinite amplitude that we had when $\omega_f = \omega_0$ for the un-damped forced oscillator.

With damping, the amplitude A has a maximum when $\omega_f \approx \omega_0$, but the amplitude remains finite.

Phase Shift: Recall that δ is the phase shift of the oscillator relative to the driving force. (we define $t = 0$ so that the driving force phase is exactly zero.)

Plot the phase shift function:



The phase shift is negative: the oscillator lags behind the forcing function. When $\omega_F \approx \omega_0$, the phase shift is about -90° .

Transient Behavior

We have been studying the long-term behavior of the forced oscillator: the motion of the oscillator once it settles into a simple repeating pattern. The long-term behavior does not depend on the initial conditions. So there are no free parameters in the long-term solution.

The initial conditions will affect the short-term behavior, however. This is called the transient

solution. Any system with non-zero damping will have its transient behavior die out as time goes forward, leaving only the long term solution. However, if the damping coefficient is small, then it may take a long time for the transient solution to go away.

How can we study the transient behavior?

The trick is to make the following observations:
Consider a forced oscillator with damping, and separately a damped oscillator with no forcing.

Forced Oscillator w/damping

Damped Oscillator

Eqs. of Motion: $\ddot{x}_F + \gamma \dot{x}_F + \omega_0^2 x_F = \frac{F_0}{m} e^{i\omega_F t}$

$$\ddot{x}_d + \gamma \dot{x}_d + \omega_0^2 x_d = 0$$

Solution: $x_F(t) = A(\omega_F) e^{i(\omega_F t + \delta_F)}$
↑ forced oscillator solution
↑ forced oscillator phase shift

$$x_d(t) = B e^{-\gamma t/2} e^{i(\omega_d t + \delta_d)}$$

↑ damped frequency
↑ damped phase shift

Number of free parameters: Zero.

Two: B and δ_d

Notice that if we add the forced solution to the damped solution, we get a new solution which also satisfies the forced equation of motion:

General Solution of forced oscillator with damping

$$x(t) = x_F(t) + x_d(t)$$

↓ forced solution
↓ damped solution

$$= A(\omega_f) e^{i(\omega_f t + \delta_f)} + B e^{-\gamma t/2} e^{i(\omega_d t + \delta_d)}$$

two free parameters (B & δ_d).

Let's show that this works:

Eq. of Motion: $\ddot{x} + \gamma \dot{x} + \omega_0^2 x \stackrel{?}{=} \frac{F_0}{m} e^{i\omega_f t}$

$$(\ddot{x}_F + \ddot{x}_d) + \gamma(\dot{x}_F + \dot{x}_d) + \omega_0^2(x_F + x_d) \stackrel{?}{=} \frac{F_0}{m} e^{i\omega_f t}$$

Gather together the damped terms:

$$\underbrace{(\ddot{x}_d + \gamma \dot{x}_d + \omega_0^2 x_d)}_{\text{damped terms}} + (\ddot{x}_F + \gamma \dot{x}_F + \omega_0^2 x_F) \stackrel{?}{=} \frac{F_0}{m} e^{i\omega_f t}$$

This equals zero according to the x_d equation of motion.

$$\ddot{X}_F + \gamma \dot{X}_F + \omega_0^2 X_F \stackrel{?}{=} \frac{F_0}{m} e^{i\omega_F t}$$

Is this true? Yes ✓, because this is the forced Eq. of Motion, and X_F is its solution.

So our general solution is:

$$X(t) = A(\omega_F) e^{i(\omega_F t + \delta_F)} + B e^{-\gamma t/2} e^{i(\omega_0 t + \delta_0)}$$

B and δ_0 should be chosen so that the initial conditions are satisfied.

Notice that the second term in the solution dies out exponentially as time goes forward due to the $e^{-\gamma t/2}$ ~~trans~~ factor. This is the transient ~~part~~ part of the solution.

After the transient dies out, we are left with the long-term behavior described by the first term in the solution.

Simple Example. Suppose we start with

$$X(t=0) = 0 \quad \text{and} \quad \dot{X}(0) = 0,$$

and suppose we drive the oscillator at the resonant frequency: $\omega_F = \omega_0$.

Also, to simplify things further, assume that damping is very small, so that

$$\omega_d = \sqrt{\omega_0^2 - \frac{\gamma^2}{4}} \approx \omega_0$$

↑

Assume
small

This is the "high-Q" assumption

Then the solution is

$$A(\omega_f) = A(\omega_0) = \frac{F_0}{m\omega_0\gamma}$$

$$\tan(\delta_f(\omega_f)) = \frac{-\omega_0\gamma}{\omega_0^2 - \omega_0^2} \rightarrow -\infty, \text{ so } \delta_f(\omega_0) = -\pi/2$$

Then the real part of the solution is:

$$x(t) = \frac{F_0}{m\omega_0\gamma} \underbrace{\cos(\omega_0 t - \pi/2)}_{\sin(\omega_0 t)} + B e^{-\gamma t/2} \cos(\omega_0 t + \delta_d)$$

$$x(t) = \frac{F_0}{m\omega_0\gamma} \sin(\omega_0 t) + B e^{-\gamma t/2} \cos(\omega_0 t + \delta_d)$$

Now determine B & δ_d from the initial conditions:

$$x(t=\phi) = \phi = B e^{-\gamma t/2} \cos(\delta_d) \Rightarrow \boxed{\delta_d = \pi/2}$$

And

$$\dot{x}(t=\phi) = \phi = \frac{F_0}{m\gamma} \cos(\omega_0 t) + B(-\omega_0 \sin(\omega_0 t + \delta_d)) e^{-\gamma t/2} - \frac{B\gamma}{2} (\cos(\omega_0 t + \delta_d)) e^{-\gamma t/2}$$

\uparrow
 $\pi/2$

At $t = \phi$:

$$\phi = \frac{F_0}{m\gamma} - B\omega_0 \underbrace{\sin(\pi/2)}_1 - \frac{B\gamma}{2} \underbrace{\cos(\pi/2)}_\phi$$

$$\phi = \frac{F_0}{m\gamma} - B\omega_0 \Rightarrow \boxed{B = \frac{F_0}{m\omega_0\gamma}}$$

Finally, when $\omega_f = \omega_0$, $\omega_d \approx \omega_0$, and $x(\phi) = \phi$,
and $\dot{x}(\phi) = \phi$,

Then

$$x(t) = \frac{F_0}{m\omega_0\gamma} \sin(\omega_0 t) + \frac{F_0}{m\omega_0\gamma} e^{-\gamma t/2} \underbrace{\cos(\omega_0 t + \frac{\pi}{2})}_{-\sin(\omega_0 t)}$$

$$\boxed{x(t) = \frac{F_0}{m\omega_0\gamma} (1 - e^{-\gamma t/2}) \sin(\omega_0 t)}$$

What does it look like?

